

Engineering Notes

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Orbital Dynamics of the Hanging Tether Interferometer

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Introduction

ASTRONOMICAL observations using long baseline optical interferometry in space are theoretically possible using numerous methods that have been examined at various levels of detail in recent years.^{1,2} One possible method that has not received significant attention is the use of a three mass gravity-gradient-stabilized tethered system in circular orbit. This system consists of two radiation collecting telescopes located at the ends of the tether and a central station (CS) which measures the coherence of the sampled beams and is located on a movable platform that moves along the tether to equalize the optical path lengths of the two beams at all times. Whether or not this will prove to be practical depends in part on whether the motions of the components of the system can be stabilized in the presence of various disturbing forces with the accuracy necessary to perform interferometry. The most important of these disturbances are the Coriolis forces associated with the movement of the central station and vibrations introduced into the system by both the tether reeling mechanism and the platform crawling mechanism. This Note examines the consequences of Coriolis forces and leaves the analysis of vibrations caused by mechanical components for a later time. A strategy for canceling the Coriolis forces using ion thrusters is analyzed and shown to use thrust levels and fuel levels that are easily within the present state of the art for a 10-km baseline interferometer operating in synchronous orbit.

U-V Plane Coverage

The geometry of the system in an operating mode is shown in Fig. 1 in which $\bar{i}, \bar{j}, \bar{k}$ is a triad of Earth-centered "inertial" unit vectors. The vector \bar{l} extends from the lower collector to the upper collector and nominally rotates at the orbital rate ω_0 around the \bar{k} direction that is perpendicular to the orbit plane. The direction of observation on the celestial sphere defines another inertial system in which \bar{k}_1 is a unit vector in the direction of observation and the unit vectors \bar{i}_1, \bar{j}_1 define the "U-V" plane in which the Fourier transform of the image is sampled by measuring the visibility and phase of fringes. Without loss of generality, \bar{i} and \bar{i}_1 may be fixed in the same direction. The angle ϕ between \bar{k} and \bar{k}_1 varies between 0 and 180 deg enabling the system to view the entire celestial sphere.

The position on the U-V plane sampled at any instant of time $\bar{D}(t)$ (called the baseline vector of the sample) is the projection of the vector \bar{l} on the U-V plane, and is given by

$$\bar{D}(t) = l [\sin(\omega_0 t) \bar{i}_1 + \cos\phi \cos(\omega_0 t) \bar{j}_1] \quad (1)$$

where l is the length of the tether. This is the locus of an ellipse for which the eccentricity is equal to $1/\cos\phi$. Thus, coverage of the U-V plane is most uniform when the source direction is perpendicular to the orbit plane, and deteriorates to a straight line in the direction parallel to the orbit plane. Coverage of a complete ellipse requires one orbital period. A complete observation of one source would require many orbits using different tether lengths, resulting in a series of concentric ellipses of arbitrary size. This is identical to the shape and the rate of coverage of the U-V plane achievable using the free-flyer approach to interferometry.³

The final sample distribution in the U-V plane is determined by the fringe sampling rate along the ellipses and the spacing between ellipses. The former is determined by the built-in system parameters (including the orbital rate) and the intensity of the source, none of which are under the immediate control of the user. The spacing between the ellipses on the other hand can be actively controlled by the user to influence the sampling distribution by varying the tether length. How this control might be utilized to achieve specific sampling objectives is the subject of another paper.⁴

Coriolis Force Disturbances

It is clear from Fig. 1 that unless $\phi = 0$ the central station (CS) must continuously move up and down the tether at the orbital rate in order to keep the optical path lengths of the two legs of the interferometer equal. The vertical motion of the CS will pull (or push) the two collectors in the opposite direction with respect to the center of mass (CM) of the entire system. If we assume that the CM is in a circular orbit, which is a reasonable approximation, then all three masses will experience Coriolis forces in the rotating coordinate system according to the well-known equation

$$\bar{F} = -2M\omega_0 \bar{V} \bar{l}_\perp \quad (2)$$

where \bar{l}_\perp is the unit vector in the orbit plane perpendicular to the tether in the direction of orbital motion, and \bar{V} is the vertical velocity, positive in the upward direction. When the CS is moving upward, it will tend to fall behind in the orbit while the two collectors that are being pulled down are tending to get ahead. A computer simulation using the tether dynamics program GTOSS with animated output has been used to verify the general appearance of the resulting motion. It clearly shows that if these forces are unopposed, they will cause large

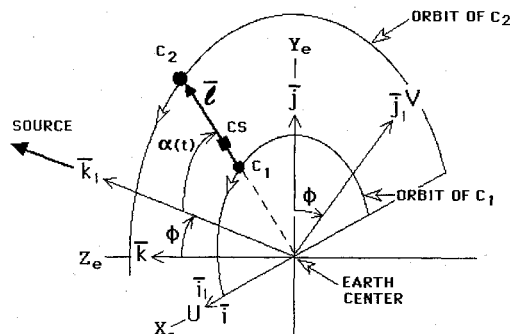


Fig. 1 Linear three-mass gravity gradient stabilized system in which the \bar{i}, \bar{j} unit vectors define the orbit plane in inertial space and \bar{i}_1, \bar{j}_1 define the U-V plane for a source in direction \bar{k}_1 on the celestial sphere.

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question can be resolved by comparing the required vertical acceleration of the collectors [found by differentiating Eq. (6)] to the vertical acceleration that is caused by gravity gradient forces. The first is easily found to be $a_c = \dot{V}_c = -K_{CS}(l/2)\omega_0^2 \sin\phi \cos(\omega_0 t)$. The gravity gradient acceleration at the ends of a two-mass dumbbell with length l is given by $a_{gg} = 3l\omega_0^2$, so that the ratio (a_{gg}/a_c) is

$$(a_{gg}/a_c) = 6/[K_{CS} \sin\phi \cos(\omega_0 t)] \quad (15)$$

Since $K_{CS} < 1$, it is clear that the gravity gradient force will always be at least six times the minimum acceptable value and will usually be much greater than that.

Conclusions

The analysis above shows that the thrust magnitude and the fuel requirements necessary for a hanging tether interferometer with a 10-km baseline are well within the present "state of the art" for a system operating in synchronous orbit, but are questionable in low Earth orbit. The thrusting necessary to oppose Coriolis forces would probably be provided by ion thrusters operating continuously using the variable thrust program also derived above. The major unanswered questions affecting the practicality of the system concern vibrations induced in the tether by the CS crawling mechanism, the reeling mechanism, and perhaps by the ion thrusters. The magnitude of these vibrations and their effect on the optical system and the attitude-control system needs to be examined both analytically and experimentally. The analytical investigation is progressing and hopefully the experimental investigation will soon follow.

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References

- ¹Stachnik, R., Melroy, P., and Arnold, D., "Multiple Spacecraft Michelson Stellar Interferometry," *SPIE Proceedings, Instrumentation in Astronomy V*, Vol. 445, 1984, p. 358.
- ²Anon., "Workshop on Optical Interferometry in Space," European Space Agency, Granada, Spain, SP-273, June 1987.
- ³DeCou, A. B., "Orbital Station Keeping for Multiple Spacecraft Interferometry," *Journal of Astronautical Sciences*, Vol. 39, No. 3, 1991, pp. 283-297.
- ⁴DeCou, A. B., "Long Baseline Optical Interferometry in Earth Orbit (System Comparisons)," *Proceedings of the SPIE 1990 Symposium on Astronomical Telescopes and Instrumentation for the 21st Century*, Feb. 11-17, 1990, pp. 626-638.

Krylov Model Reduction Algorithm for Undamped Structural Dynamics Systems

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Introduction

KRYLOV vectors have been shown to form an efficient basis for eigenvalue analysis and model reduction of structural dynamics systems.¹⁻³ In Ref. 4, a Krylov model re-

duction algorithm is developed for a damped structural dynamics system described by a matrix second-order differential equation together with an output measurement equation. Numerical results have shown that Krylov-based reduced models have some advantages over normal mode reduced models in the application to structural control problems. The purpose of this Note is to present a Krylov model reduction algorithm for undamped systems. The undamped case covered in this Note supplements the damped case in Ref. 4, completing the work on Krylov model reduction methods.

The Krylov model reduction algorithm presented here has a feature similar to the Lanczos algorithm in Ref. 2. However, it is a more general case because the algorithm includes the output measurement equation. It is also shown that the Krylov model reduction method has an association with the well-known parameter-matching method for general linear systems.^{5,6} The reduced-order model obtained by the Krylov model reduction algorithm proposed here is shown to match a set of parameters called low-frequency moments. For a general linear time-invariant system described by

$$\dot{z} = Az + Bu \quad z \in R^n, u \in R^l \quad (1a)$$

$$y = Cz \quad y \in R^m \quad (1b)$$

the low-frequency moments are defined by $CA^{-i}B$, $i = 1, 2, \dots$, which are the coefficient matrices in the Taylor series expansion of the system transfer function. One advantage of using Krylov reduced models for structural control design is that the Krylov formulation can eliminate the control and observation spillover terms while leaving only the dynamic spillover terms to be considered.⁴

In this Note, first a theorem is used to show how the Krylov vectors can form a basis to produce a reduced model with moment-matching property. Then, based on the theorem, an efficient algorithm to generate Krylov vectors is developed.

Undamped Structural Dynamics Systems

An undamped structural dynamics system can be described by the input-output equations

$$M\ddot{x} + Kx = Pu \quad (2a)$$

$$y = Vx + W\dot{x} \quad (2b)$$

where $x \in R^n$ is the displacement vector; $u \in R^l$ is the input force vector; $y \in R^m$ is the output measurement vector; M and K are the system mass and stiffness matrices, respectively; P is the force distribution matrix; and V and W are the displacement and velocity sensor distribution matrices, respectively. In most practical cases, it may be assumed that l and m are much smaller than n .

Applying the Fourier transform to Eq. (2a) yields the frequency response solution $X(\omega) = (K - \omega^2 M)^{-1} P U(\omega)$, with $X(\omega)$ and $U(\omega)$ the Fourier transforms of x and u . If the system is assumed to have no rigid-body motion, then a Taylor series expansion of the frequency response around $\omega = 0$ is possible. Thus,

$$\begin{aligned} X(\omega) &= (I - \omega^2 K^{-1} M)^{-1} K^{-1} P U(\omega) \\ &= \sum_{i=0}^{\infty} \omega^{2i} (K^{-1} M)^i K^{-1} P U(\omega) \end{aligned} \quad (3)$$

Combining Eq. (2b) and Eq. (3), the system output frequency response can be expressed as

$$\begin{aligned} Y(\omega) &= \sum_{i=0}^{\infty} [V(K^{-1} M)^i K^{-1} P \\ &\quad + j\omega W(K^{-1} M)^i K^{-1} P] \omega^{2i} U(\omega) \end{aligned} \quad (4)$$

In these expressions, $V(K^{-1} M)^i K^{-1} P$ and $W(K^{-1} M)^i K^{-1} P$ play roles similar to that of low-frequency moments in the first-order state-space formulation. Therefore, we have the following definition.

Definition 1. The low-frequency moments of an undamped structural dynamics system described by Eqs. (2) are

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