Liquid Phase Enthalpy Values for the Methane-Ethane System

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IN PREVIOUS investigations (8, 10, 11) saturated liquid phase enthalpies have been calculated for five binary hydrocarbon systems over the pressure range 100 to 600 p.s.i.a. These enthalpy data were calculated from P-V-T-x data and the differential heat of condensation. The basic relationships, which are thermodynamically rigorous, employed are given by Dodge (7) and are

$$\left(\frac{\partial P}{\partial T}\right)_{y} = \frac{\Delta H_{c}}{T\Delta V_{c}}$$
(1)

$$\Delta V_c = V_b - V_d + (y - x) \left(\frac{\partial V_G}{\partial y}\right)_{T, P}$$
⁽²⁾

$$\Delta H_c = H_b - H_d + (y - x) \left(\frac{\partial H_G}{\partial y} \right)_{T, P}$$
(3)

The Benedict-Webb-Rubin (3, 4) equation of state

 $P = RTd + (B_o RT - A_o - C_o / T^2)d^2 + (b' RT2a')d^3$

$$+ a' \alpha d^{6} + c d^{3} / T^{2} [(1 + \gamma d^{2}) \exp(-\gamma d^{2})]$$

$$\tag{4}$$

was checked against the available vapor phase volume for the various hydrocarbon systems and found useful. Usually the difference between the experimental and the predicted values was less than $\pm 1\%$. The Benedict-Webb-Rubin equation was then used to calculate additional volumetric data and the necessary partial volumes were obtained by graphical differentiation.

The form of the Benedict-Webb-Rubin equation

$$H - \sum_{i} x_{i} H_{i}^{\circ} = (B_{\circ} RT - 2A_{\circ} - 4C_{\circ}/T^{2})d$$

$$+ (2b'RT - 3a')d^{3}/_{2} + 6a'\alpha d^{5}/_{5} + cd^{2}/T^{2}$$

$$\left[3 \frac{1 - \exp(-\gamma d^{2})}{^{2}d^{2}} - \frac{\exp(-\gamma d^{2})}{2} + \exp(\gamma d^{2}) \right]$$
(5)

in conjunction with ideal gas state enthalpies obtained from API Research Project 44 (1) were used to predict total enthalpy values for gas mixtures. Partial enthalpy values were then obtained by graphical differentiation. With the calculated enthalpy and volumetric data and experimental vapor-liquid equilibrium data, and information on the volumetric behavior of the saturated liquid phase, the differential heat of vaporization and, in turn, saturated liquid phase enthalpies could be calculated by Equations 1, 2, and 3.

While liquid phase enthalpy data of saturated mixtures are useful and of interest, partial values—in a sense—are more useful because the contribution of each component to the total enthalpy of the mixture is known. With this thought in mind this work was undertaken to evaluate partial enthalpies in the liquid phase for the methaneethane system. This binary system was selected because it is of some industrial interest and extensive P-V-T-x data have been determined by Bloomer, Gami, and Parent (6).

One method of obtaining partial enthalpy values is to calculate saturated enthalpies by Equations 1, 2, and 3 at a number of different pressures and, if the effect of pressure on the liquid phase enthalpy is negligible, which is certainly a valid assumption if the pressure range is small, then draw in various isotherms on a plot of enthalpy vs. composition. From the slope of an isotherm at a given point, the partial enthalpy values of the components of the particular mixture at the specific conditions of temperature and pressure can be evaluated.

An attempt was made to determine partial enthalpies by this procedure. Saturated liquid enthalpies were calculated at intervals of 50 p.s.i. over the pressure range of 100 to 700 p.s.i.a. The results were not consistent and clearly partial enthalpies could not be estimated. While the method employed is thermodynamically rigorous, certain difficulties arise when attempts are made to evaluate some of the required terms.

Since the results obtained using the differential heat of condensation were unsatisfactory, it was decided to work with the differential heat of vaporization. A similar type of reasoning used to develop Equations 1, 2, and 3 for a condensation process can be applied to a vaporization process and an analogous set of equations developed. The equations in the latter case would be

$$\left(\frac{\partial P}{\partial T}\right)_{x} = \frac{\Delta H_{v}}{T\Delta V_{v}} \tag{6}$$

$$\Delta V_{c} = V_{d} - V_{b} - (y - x) \left(\frac{\partial V_{L}}{\partial x}\right)_{T_{c}P}$$
(7)

$$\Delta H_{v} = H_{d} - H_{b} - (y - x) \left(\frac{\partial H_{L}}{\partial x}\right)_{T, P}$$
(8)

 ΔH_v represents the differential heat of vaporization, which is the heat required to vaporize a mole of mixture of composition y_1 from a large quantity of liquid of composition x_1 . The composition of the liquid remains unchanged in the process and the relationship between y_1 and x_1 is the equilibrium condition. With the experimental data available, the effect of pressure on the liquid volume over a 50-p.s.i. pressure differential had to be neglected. This effect is undoubtedly negligible. In other words, in evaluating ΔV_v by Equation 7, $(\partial V_L/\partial x)_T$ was used in place of $(\partial V_L/\partial x)_T p$. With this value and the experimental data, the differential heat of vaporization could be determined from Equation 6.

If the original value of the liquid phase enthalpy as calculated by Equations 1, 2, and 3 had been correct, the term $(\partial H_L/\partial x)_{T,P}$ could have been calculated directly from Equation 8 and no further work would have been necessary. At pressures of 450 and 500 p.s.i.a. this type of straightforward solution was obtained. To show how well the procedure worked in these instances, plots of ΔH_c vs. x and ΔH_v vs. x at 500 p.s.i.a. are included as Figures 1 and 2, respectively. The differential heats calculated by Equations 1 and 6 are compared graphically with those obtained from Equations 3 and 8, in which the final results were substituted.

This was not the situation in the majority of the cases, however, and adjustments had to be made in the values of the saturated liquid phase enthalpies, H_b 's. Any alteration in the value of H_b meant a change of the same magnitude in $(y - x)(\partial H_L/\partial x)_T$, specifically the second term of the product, because Equation 6 must be satisfied. Naturally, in doing this Equation 3 was no longer satisfied. A trial and error procedure was then employed to obtain a set of values of H_b 's and $(y - x) (\partial H_L / \partial x)_T$ which was consistent with Equations 6, 7, and 8.

Basically, the assumption was made that the differential heat of vaporization was more accurate than the differential heat of condensation. The reasons for assuming this were: (1) $(\partial P/\partial T)_x$ showed less variation than $(\partial P/\partial T)_y$. The variations in the latter quantity were particularly large at high methane concentrations. (2) Partial volumes and partial enthalpies in the gas phase were obtained from an equation of state; consequently, these values are subject to error. Contrariwise, the partial volumes in the liquid phase could be obtained from the available experimental data, provided the effect of an additional pressure of 50 p.s.i. was neglected. (3) Partial enthalpies in the liquid phase were, in essence, the final answer and were not used in the calculation. This was not true of partial enthalpies in the gas phase. These gas phase partial enthalpies may well be a source of error. (4) ΔV_v was frequently approximately the magnitude of V_d . This was not true of the relationship between ΔV_c and V_d , since partial gas phase volumes were much larger than partial liquid phase volumes. Dew point volumes are accurate, as these values were obtained by the Benedict equation of state which was checked against experimental values and modified by Bloomer and others (6) to obtain good agreement in this area. One would conclude that ΔV_v tended to be more accurate than ΔV_c .

Usually the differential heat of condensation appeared to be too large by from 200 to 400 B.t.u.'s per pound mole. Only in a few instances did the differential heat of condensation appear to be too small. A plot of ΔH_c vs. x at a pressure of 650 p.s.i.a is included as Figure 3 to show the deviation between the calculated values and the values finally used. The maximum difference in this particular instance was 400 B.t.u. per lb. mole; hence, the illustration reflects the maximum deviation. For comparison a plot (Figure 4) of ΔH_c vs. x at 650 p.s.i.a is also included. Here the differences between the calculated and final values are small.

Only at pressures of 100 and 150 p.s.i.a. did the ΔH_c values appear to be more accurate than the ΔH_c 's. Here the properties of the gaseous mixture tended to be the direct combination of the properties of the pure components.

As a further check on the gas phase volumes and partial volumes used in the calculations, these values at 70 mole % methane were determined using the relationship presented by Black (5).

$$V_{m} = \frac{RT}{P} + \sum b_{i} y_{i} - \frac{\left[\sum (a_{i} \zeta^{*}_{i})^{0.5} y_{i}\right]^{2}}{RT}$$
(9)

The volumes calculated by Equation 9 were in good agreement with the experimental values and those determined by the Benedict-Webb-Rubin equation. However, the partial volumes determined by the relationship





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Figure 2. Calculated and adjusted ΔH_{ν} values

$$V_{i} = \frac{RT}{P} + b_{i} - \frac{a_{i}\zeta_{i}}{RT} + \frac{(\sum_{j=1}^{r} G_{ij}y_{j})^{2}}{RT}$$
(10)

also given by Black, differed by $\pm 12\%$ from those obtained by the Benedict-Webb-Rubin equation. This points out the difficulties involved in calculating partial volumes from an equation of state. The equation may predict the total volume reasonably well, but be in error in the partial volume. To determine whether the partial volumes predicted by one equation of state are more accurate than those predicted by another equation would be rather difficult.

The final liquid phase enthalpy data are illustrated on Figure 5 and Table I. The enthalpies presented differ from those given previously (10). The former values were obtained by use of the differential heat of condensation, while the values presented here are based chiefly on the differential heat of vaporization. For the reasons mentioned, the new values are believed to be the more accurate.

Also given in Table I are bubble point and dew point temperatures, dew point enthalpies, and partial enthalpies for the liquid phase. The pure component data are those of Matthews and Hurd (9) for methane and of Barkelew, Valentine, and Hurd (2) for ethane. The partial enthalpies and ethane are shown in Figures 6 and 7, respectively. For comparison the saturation curves for the pure substances are included on each figure. While quantitative conclusions cannot be reached, composition, temperature, and pressure are clearly important parameters in the evaluation of partial enthalpy values.

"Ideal" enthalpy values, assuming additive properties and estimating the effect of pressure on the enthalpy of the pure components, of the saturated liquid were calculated for pressures of 100 and 200 p.s.i.a. over the composition ranges for which the bubble point temperature of the mixture did not exceed the critical temperature of methane. In Figure 8, these values are compared with the enthalpies of the real mixture calculated in this work. The comparisons indicate the integral heat of solution per mole of mixture is relatively small. This is to be expected in a mixture of paraffin hydrocarbons at conditions which are not severe.

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NOMENCLATURE

$$H =$$
 enthalpy, B.t.u. per lb. mole

 ΔH_c = differential heat of condensation, B.t.u. per lb. mole



Figure 3. Calculated and adjusted ΔH_c values

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Figure 4. Calculated and adjusted ΔH_{ν} values

Table I. Enthalpy and Partial Enthalpy Values for Saturated Mixtures of Methane and Ethane^a

Mole Fraction	Temp	o., ° F.	Enthalpy, B.t.u./Lb. Mole							
Methane	t,	t _d	H_b	H_b H_d		H_2				
Pressure = 100 p.s.i.a.										
0	·-46.0	-46.0	-2060	3299		-2061(2)				
0.10	-101.7	-52.2	-2765	3242	+118	-3085				
0.20	-136.3	-58.5	-3048	3182	-495	-3686				
0.30	-156.0	-65.0	-3074	3112	-743	-4073				
0.40	-171.2	-72.7	-2982	3031	-887	-4379				
0.50	-181.1	-80.9	-2795	2933	-988	-4602				
0.60	-187.5	-90.7	-2537	2848	-1065	-4745				
0.70	-192.8	-102.2	-2235	2735	-1125	-4825				
0.80	-197.3	-117.7	-1914	2598	-1173	-4878				
0.90	-201.6	-140.0	-1581	2396	-1210	-4920				
1.00	-205.5	-205.5	-1244	1812	-1244	(9)				

(Continued on next page)



Table	I.	(Continued)
10010	•••	(00	1

Mole Fraction	Temp	o., ° F	Ent	halpy, B	.t.u./Lb.	Mole	Mole Fraction	Temp	o., ° F	Entl	halpy, B	.t.u./Lb.	Mole
Methane	t_b	t _d	H_{\flat}	H_{d}	H_1	H_2	Methane	t _b	td	H_b	H_d	H_1	H_2
Pressure = 150 p.s.i.a.									Pressure	e = 450 p.	.s.i.a.		
$\begin{matrix} 0 \\ 0.10 \\ 0.20 \\ 0.30 \\ 0.40 \\ 0.50 \\ 0.60 \\ 0.70 \\ 0.80 \\ 0.90 \\ 1.00 \end{matrix}$	$\begin{array}{r} -24.2 \\ -74.0 \\ -106.6 \\ -131.1 \\ -149.5 \\ -161.0 \\ -169.0 \\ -175.4 \\ -180.6 \\ -185.6 \\ -190.0 \end{array}$	$\begin{array}{r} -24.2 \\ -30.2 \\ -37.4 \\ -44.9 \\ -53.5 \\ -62.2 \\ -73.2 \\ -87.0 \\ -103.0 \\ -127.7 \\ -190.0 \end{array}$	$\begin{array}{r} -1630 \\ -2279 \\ -2540 \\ -2658 \\ -2654 \\ -2487 \\ -2246 \\ -1958 \\ -1650 \\ -1330 \\ -1002 \end{array}$	3400 3339 3274 3202 3121 3028 2920 2795 2646 2428 1852	570 + 22 - 418 - 637 - 760 - 821 - 900 - 947 - 979 - 1002	$\begin{array}{c} -1630(2) \\ -2596 \\ -3181 \\ -3618 \\ -3999 \\ -4214 \\ -4384 \\ -4427 \\ -4462 \\ -4489 \\ (9) \end{array}$	$\begin{array}{c} 0\\ 0.10\\ 0.20\\ 0.30\\ 0.40\\ 0.50\\ 0.60\\ 0.70\\ 0.80\\ 0.90\\ 1.00\\ \end{array}$	52.3 + 18.0 - 13.3 - 43.1 - 67.5 - 87.0 - 140.0 - 112.5 - 121.1 - 130.0 - 138.8	$52.3 \\ 41.5 \\ 30.9 \\ 20.1 \\ +7.7 \\ -5.7 \\ -21.7 \\ -39.0 \\ -60.6 \\ -90.0 \\ -138.8 \\ -90.0 \\ -138.8 \\ -90.0 \\ -138.8 \\ -90.0 \\ -138.8 \\ -90.0 \\ -138.8 \\ -90.0 \\ -138.8 \\ -90.0 \\ -138.8 \\ -90.0 \\ -138.8 \\ -90.0 \\ -9$	$\begin{array}{r} +119 \\ -380 \\ -800 \\ -1060 \\ -1152 \\ -1159 \\ -1081 \\ -922 \\ -701 \\ -419 \\ -98 \end{array}$	3458 3410 3358 3290 3205 3081 2953 2800 2615 2331 1763	2575 2000 1464 1020 655 433 298 185 +60 -98	$\begin{array}{c} +119(2) \\ -708 \\ -1500 \\ -2142 \\ -2600 \\ -2973 \\ -3352 \\ -3769 \\ -4245 \\ -4730 \\ (9) \end{array}$
		Pressure	e = 200 p.	s.i.a.				Pressure = 500 p.s.i.a.					
$\begin{matrix} 0 \\ 0.10 \\ 0.20 \\ 0.30 \\ 0.40 \\ 0.50 \\ 0.60 \\ 0.70 \\ 0.80 \\ 0.90 \\ 1.00 \end{matrix}$	$\begin{array}{r} -6.2 \\ -51.0 \\ -85.5 \\ -112.0 \\ -130.0 \\ -144.0 \\ -154.0 \\ -161.2 \\ -167.1 \\ -172.7 \\ -178.5 \end{array}$	$\begin{array}{r} -6.2 \\ -13.4 \\ -20.9 \\ -29.0 \\ -37.9 \\ -48.5 \\ -60.9 \\ -75.0 \\ -92.2 \\ -117.7 \\ -178.5 \end{array}$	-1231 -1850 -2181 -2323 -2327 -2222 -2015 -1785 -1490 -1163 -811	3467 3413 3344 3263 3170 3069 2952 2820 2661 2423 1866	$\begin{array}{r} 850 \\ +455 \\ -90 \\ -388 \\ -534 \\ -578 \\ -656 \\ -700 \\ -750 \\ -811 \end{array}$	$\begin{array}{c} -1231(2) \\ -2150 \\ -2840 \\ -3280 \\ -3620 \\ -3910 \\ -4170 \\ -4420 \\ -4650 \\ -4876 \\ (9) \end{array}$	$\begin{array}{c} 0\\ 0.10\\ 0.20\\ 0.30\\ 0.40\\ 0.50\\ 0.60\\ 0.70\\ 0.80\\ 0.90\\ 1.00 \end{array}$	$\begin{array}{r} 61.0\\+28.0\\-3.8\\-32.2\\-57.3\\-77.1\\-92.4\\-104.6\\-114.1\\-123.5\\-133.1\end{array}$	$\begin{array}{c} 61.0\\ 49.4\\ 38.3\\ 26.9\\ +13.9\\ 0.0\\ -16.4\\ -34.3\\ -56.4\\ -86.6\\ -133.1\end{array}$	+364 -141 -573 -856 -957 -966 -893 -749 -540 -273 +30	3398 3350 3302 3249 3163 3050 2916 2768 2587 2291 1715	$\begin{array}{c} 3270\\ 2335\\ 1767\\ 1373\\ 908\\ 622\\ 475\\ 375\\ 241\\ 30\end{array}$	+364(2) -520 -1300 -2510 -2840 -3165 -3605 -4200 -4897 (9)
0		Pressure	e = 250 p.	s.i.a.		001(0)	0	6 0 0	Pressure	e = 550 p.	s.i.a.		. 005 (0)
0 0.10 0.20 0.30 0.40 0.50 0.60 0.70 0.80 0.90 1.00	$\begin{array}{r} +8.8 \\ -34.0 \\ -68.3 \\ -95.4 \\ -115.3 \\ -130.0 \\ -141.4 \\ -149.0 \\ -155.6 \\ -162.1 \\ -168.2 \end{array}$	$\begin{array}{r} +8.8 \\ +1.9 \\ -7.4 \\ -16.2 \\ -25.9 \\ -37.1 \\ -50.5 \\ -65.1 \\ -83.6 \\ -109.8 \\ -168.2 \end{array}$	-901 -1530 -1841 -2004 -2050 -1992 -1852 -1617 -1323 -999 -652	3545 3468 3387 3301 3207 3102 2984 2845 2676 2451 1868	$\begin{array}{r} 1400 \\ 715 \\ +308 \\ -93 \\ -334 \\ -473 \\ -531 \\ -561 \\ -601 \\ -652 \end{array}$	$\begin{array}{r} -901(2) \\ -1855 \\ -2480 \\ -2995 \\ -3355 \\ -3650 \\ -3920 \\ -4150 \\ -4370 \\ -4580 \\ (9) \end{array}$	$\begin{array}{c} 0 \\ 0.10 \\ 0.20 \\ 0.30 \\ 0.40 \\ 0.50 \\ 0.60 \\ 0.70 \\ 0.80 \\ 0.90 \\ 1.00 \end{array}$	$\begin{array}{r} 68.9\\ 37.2\\ +6.8\\ -22.0\\ -47.5\\ -68.3\\ -96.7\\ -107.4\\ -117.4\\ -127.5\end{array}$	68.9 56.4 44.8 32.9 19.7 +5.2 -11.9 -30.2 -52.4 -82.9 -127.5	$\begin{array}{r} 605 \\ +118 \\ -312 \\ -638 \\ -760 \\ -784 \\ -720 \\ -580 \\ -378 \\ -112 \\ +195 \end{array}$	3307 3282 3253 3212 3140 3016 2882 2727 2541 2238 1655	3700 2740 2027 1640 1142 870 727 570 420 195	+605(2) -280 -1075 -2360 -2710 -3105 -3630 -4170 -4900 (9)
	_	Pressure	e = 300 p.	s.i.a.					Pressure	= 600 p.s	s.i.a.		
0 0.10 0.20 0.30 0.40 0.50 0.60 0.70 0.80 0.90 1.00	$\begin{array}{r} +21.6 \\ -18.2 \\ -52.6 \\ -80.2 \\ -102.4 \\ -118.0 \\ -129.9 \\ -138.6 \\ -145.9 \\ -152.9 \\ -159.2 \end{array}$	$\begin{array}{c} 21.6\\ 13.1\\ +4.2\\ -5.2\\ -15.7\\ -27.5\\ -41.2\\ -56.9\\ -96.3\\ -103.8\\ -159.2\end{array}$	$\begin{array}{r} -650 \\ -1238 \\ -1578 \\ -1728 \\ -1802 \\ -1782 \\ -1652 \\ -1450 \\ -1180 \\ -859 \\ -502 \end{array}$	3515 3459 3398 3321 3327 3118 2994 2850 2681 2432 1853	$1319 \\ 1040 \\ 650 \\ +246 \\ -62 \\ -240 \\ -345 \\ -415 \\ -467 \\ -502$	$\begin{array}{r} -650(2) \\ -1522 \\ -2233 \\ -2747 \\ -3167 \\ -3502 \\ -3770 \\ -4028 \\ -4240 \\ -4387 \\ (9) \end{array}$	$\begin{array}{c} 0 \\ 0.10 \\ 0.20 \\ 0.30 \\ 0.40 \\ 0.50 \\ 0.60 \\ 0.70 \\ 0.80 \\ 0.90 \\ 1.00 \end{array}$	$\begin{array}{c} 75.8\\ 46.5\\ +16.6\\ -12.2\\ -37.0\\ -59.5\\ -77.0\\ -90.0\\ -101.0\\ -111.7\\ -122.7\end{array}$	75.8 62.9 50.7 38.2 24.6 +9.9 -7.6 -26.7 -49.6 -80.5 -122.7	$\begin{array}{r} 870 \\ +380 \\ -70 \\ -413 \\ -550 \\ -605 \\ -561 \\ -401 \\ -183 \\ +72 \\ 352 \end{array}$	3195 3191 3172 3127 3045 2931 2801 2656 2466 2176 1543	4260 3030 2380 1860 1415 1120 1000 980 852 352	$\begin{array}{r} +870(2) \\ -51 \\ -845 \\ -1610 \\ -2157 \\ -2625 \\ -3083 \\ -3670 \\ -4835 \\ -6948 \\ (9) \end{array}$
Pressure = 350 p.s.i.a.						0	90.0	Pressure	e = 650 p.s	s.i.a.		1000(0)	
0.10 0.20 0.30 0.40 0.50 0.60 0.70 0.80 0.90 1.00	-32.9 -5.6 -38.1 -67.2 -90.0 -107.0 -119.8 -129.0 -135.2 -144.6 -151.8	22.5 23.5 14.0 +4.2 -7.0 -19.2 -33.3 -49.9 -70.4 -98.1 -151.8 Pressure	-400 -971 -1298 -1500 -1558 -1542 -1453 -1282 -1041 -728 -370 e = 400 p	3453 3472 3318 3221 3113 2990 2844 2658 2408 1830 s.i.a.	$1860 \\ 1350 \\ 893 \\ 496 \\ +185 \\ 0 \\ -125 \\ -225 \\ -305 \\ -370 $	-1286 -1960 -2526 -2927 -3269 -3982 -4305 -4535 (9)	$\begin{array}{c} 0 \\ 0.10 \\ 0.20 \\ 0.30 \\ 0.40 \\ 0.50 \\ 0.60 \\ 0.70 \\ 0.80 \\ 0.90 \\ 1.00 \end{array}$	52.2 54.6 +25.2 -3.0 -28.3 -51.5 -70.2 -83.0 -93.9 -105.6 -118.0	62.2 68.8 55.7 42.9 28.9 +13.8 -4.0 -23.6 -46.8 -78.5 -118.0 Pressure	$\begin{array}{r} 1236\\ 659\\ +149\\ -228\\ -383\\ -432\\ -419\\ -263\\ +5\\ +386\\ 828\\ \end{array}$	3028 3103 3118 3080 3007 2915 2792 2625 2429 2102 1468 s.i.e.	$5260 \\ 3300 \\ 2555 \\ 2050 \\ 1620 \\ 1370 \\ 1140 \\ 1000 \\ 900 \\ 828$	$\begin{array}{c} 1236(2) \\ +148 \\ -639 \\ -1421 \\ -2005 \\ -2484 \\ -3103 \\ -3538 \\ -3975 \\ -4240 \\ (9) \end{array}$
0	43.0	43.0	– 400 p. – 135	3495		-135(2)	0	88.3	- 1055ure 88.3	1748 1748	2620		1748(2)
$\begin{array}{c} 0.10\\ 0.20\\ 0.30\\ 0.40\\ 0.50\\ 0.60\\ 0.70\\ 0.80\\ 0.90\\ 1.00 \end{array}$	$\begin{array}{r} +6.6\\ -27.0\\ -55.0\\ -79.2\\ -96.7\\ -109.7\\ -120.0\\ -129.1\\ -137.2\\ -145.2\end{array}$	$\begin{array}{r} 33.2\\ 23.4\\ 12.6\\ +0.4\\ -11.8\\ -26.8\\ -44.1\\ -64.8\\ -93.7\\ -145.2\end{array}$	$\begin{array}{r} -687 \\ -1092 \\ -1288 \\ -1353 \\ -1337 \\ -1253 \\ -1100 \\ -877 \\ -574 \\ -237 \end{array}$	3456 3389 3305 3203 3092 2960 2818 2639 2392 1803	$2167 \\ 1625 \\ 1141 \\ 738 \\ 420 \\ 250 \\ +100 \\ -27 \\ -138 \\ -237$	$\begin{array}{c} -1004\\ -1771\\ -2329\\ -2747\\ -3094\\ -3508\\ -3900\\ -4277\\ -4498\\ (9)\end{array}$	0.10 0.20 0.30 0.40 0.50 0.60 0.70 0.80 0.90 0.9825	$\begin{array}{r} 62.5\\ 34.2\\ +6.0\\ -19.9\\ -43.2\\ -62.5\\ -76.5\\ -88.2\\ -100.0\\ -110.4\end{array}$	73.660.647.233.1+17.4-1.1-20.9-44.6-76.5-110.4*	$\begin{array}{r} 993\\ 400\\ +20\\ -198\\ -250\\ -250\\ -130\\ +139\\ 607\\ 1312 \end{array}$	2971 3002 2993 2949 2853 2727 2563 2369 1991 1312	7491 4360 3053 2370 1951 1511 1310 1336 1819	+271 -590 -1280 -1910 -2451 -3490 -4649 -10,301

 $^aH=0$ for pure components in ideal gas state, at unit fugacity, and 0° R. Vapor and liquid phase enthalpies of pure components

obtained from literature sources cited were all adjusted to this state. $^{\rm b}$ Critical point.



Figure 6. Partial enthalpies of ethane Pure component data of Matthews and Hard (9). H = 0 for pure methane in ideal gas state, unit fugacity, and 0° R.



Figure 7. Partial enthalpies of ethane Pure component data of Barkelew, Valentine, and Hurd (2) H = 0 for pure ethane in ideal gas state, unit fugocity, and 0° R.

- ΔH_v = differential heat of vaporization, B.t.u. per lb. mole
 - Р = pressure, p.s.i.a.
 - R = gas law constant, 10.731 (p.s.i.) (cu. ft.) per (lb. mole) (° R.)
 - Ttemperature, ° R. =
 - VŦ volume, cu. ft. per lb. mole
- $\Delta V_c =$ volume change accompanying differential condensation process, cu. ft. per lb. mole
- $\Delta V_{\rm m} =$ volume change accompanying differential vaporization process, cu. ft. per lb. mole
 - V partial volume, cu. ft. per lb. mole



Figure 8. Ideal and actual saturated liquid phase enthalpies

- a Van der Waals attraction constant
- Van der Waals constant, covolume b =
- density, lb. moles per cu. ft. d =
- mole fraction in liquid phase x =
- y = mole fraction in vapor phase
- G, ζ = quantities in Black's equation of state

 $A_o, B_o, C_o, a', b', c, \alpha, \beta$ = empirical constants of Benedict-Webb-Rubin equation of state

Subscripts

- G =gas phase
- L = liquid phase
- b =bubble point
- d =dew point
- i = component i in a mixture
- = mixture m l =
 - component l in a mixture

Superscript

o = property in ideal gas state

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