

**THEORETICAL INVESTIGATIONS OF THE HEAT TRANSFER AT QUANTITATIVE DSC-MEASUREMENTS AND ITS INFLUENCE ON THE DETERMINATION OF THE THERMAL AND CALORIMETRICAL PROPERTIES**

W. Poeßnecker

Fraunhofer-Einrichtung für Keramische Technologien und Sinterwerkstoffe, Dresden<sup>1)</sup>  
Helmholtzstr. 20  
0 - 8027 Dresden

**ABSTRACT**

In this work the deviations of measured calorimetical DSC-signals and transition temperatures from their true values are mathematically calculated by exact analytical and numerical methods.

The obtained results can be used not only for a better understanding of the procedure of measurement but also for minimizing deviations.

Especially in the case of thermal deviations two procedures for obtaining the true transition temperatures are recommended.

**1. INTRODUCTION**

In a previous work of the author a theory of heat transfer at DSC measurements was developed [1], but this theory was related only to specific heat. In this work the influence of heat transfer on the phase transitions and transition temperatures will be taken into account, too.

1) Institut für Pulvermetallurgie und Verbundwerkstoffe,  
8027 Dresden, Helmholtzstraße 20

## 2. PHYSICAL MODEL

### 2.1. Sample holder system

In this work an usual disk-shaped arrangement of the sample and sample holder is used [2], which is shown schematically at figure 1.

The pan and the sample are substituted by disks and the heat transfer coefficients have following meaning.

$\alpha_{1R}$  = heat transfer coefficient between sample holder and pan

$\alpha_{3R}$  = heat transfer coefficient between pan and sample

$\alpha_{2R}$  = heat transfer coefficient between sample and the surroundings

( $\alpha_{1R}$  and  $\alpha_{3R}$  are inversely proportional to contact resistances,  $\alpha_{2R}$  is proportional to heat dissipation to surroundings).

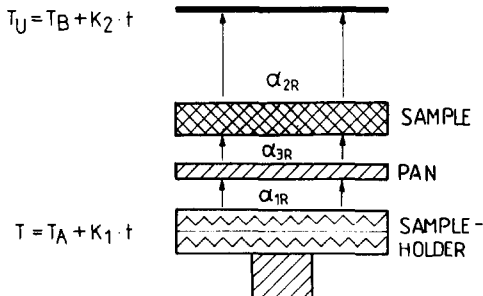


Fig. 1 Calculation model for the influence of the heat-transfer coefficients

The sample holder is heated according to

$$T = T_A + K_1 t$$

where

$$\begin{aligned} T_A &= \text{constant initial temperature of the sample holder} \\ K_1 &= \text{heating rate} \\ t &= \text{time} \end{aligned}$$

Thus heat flows from the sample holder through the pan and the sample to the surroundings with the temperature  $T_u$ .

This heat flux depends not only on the mentioned heat-transfer coefficients but also on heating rate, density, thickness, specific heat and heating conductivity of the pan and sample and on the specific transition heat if a phase transition is present. All these dependencies will be calculated in this work.

In the case of heat-conducting DSC the temperature  $T_u$  of surroundings is with  $K_2 = K_1$  (see fig. 1)

$$T_u = T_B + K_1 \cdot t$$

and in the case of power compensated DSC the temperature  $T_u$  of the isoperibole surroundings is

$$T_u = T_B = \text{const},$$

where in the case of the heat conducting DSC  $T_B$  is the constant initial temperature of surroundings and in the case of power compensated DSC the permanent constant temperature of surroundings.

This simplified one-dimensional arrangement is sufficient enough for a good thermophysical characterisation of the considered system.

## 2.2. REFERENCE SYSTEM

For increasing the sensitivity a reference system on the left side with the same capacity as the empty sample system on the right side is heated always in the same manner.

The sample system on the right side has the index R and the reference system on the left side has the index L. Thus the difference of heating fluxes in the right and left system is measured in the both DSC-methods.

For reducing the calculation effort in this work the power compensated DSC will be considered only.

### 2.3. ZERO SETTING PROCEDURE FOR DIFFERENTIAL SIGNAL OF HEAT FLUX

For empty pans in the sample system and in the left system the differential heat flux  $\Delta W_o$  depends on the heat transfer coefficients  $\alpha_{1L}$ ,  $\alpha_{2L}$ ,  $\alpha_{1R}$ ,  $\alpha_{2R}$ ,  $\alpha_{3R}$  in a complicated manner.

For simplifying the problem the following procedure is used (see fig. 2). In a first step a measurement without the sample is accomplished. By using slope correction and zero setting the differential signal  $\Delta W_o \cong 0$  is obtained.

In a second step a sample is inserted into the right system and the measurement of the first step is repeated under the same conditions. Then the obtained signal  $\Delta W = W_p$  is representative for the pure thermophysical properties of sample only, that means the very complicated dependence of the differential signal on the heat transfer coefficients  $\alpha_{1L}$ ,  $\alpha_{2L}$ ,  $\alpha_{1R}$  and  $\alpha_{2R}$  is compensated if the values of  $\alpha_{1R}$  and  $\alpha_{2R}$  are exactly the same as in the first step. For the values of  $\alpha_{1L}$ ,  $\alpha_{2L}$  and  $\alpha_{1R}$  there is no problem because it is not necessary to alter the left and right system of empty pans before the second step.

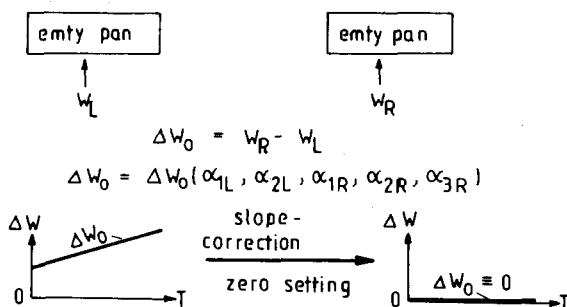
After the second step complications may arise not only from the heat-transfer coefficient  $\alpha_{3R}$  but also from a possible deviation of the value of  $\alpha_{2R}$  between the first and second step.

If  $(\alpha_{2R})(1)$  is the heat transfer coefficient  $\alpha_{2R}$  at the first step and  $(\alpha_{2R})(2)$  is the heat transfer coefficient  $\alpha_{2R}$  at the second step then a new effective heat transfer coefficient can be defined as

$$\alpha_{2e} = (\alpha_{2R})(2) - (\alpha_{2R})(1)$$

which has to be taken into account after sloping correction and zero setting procedure in the first step.

first step: (1)



second step: (2)

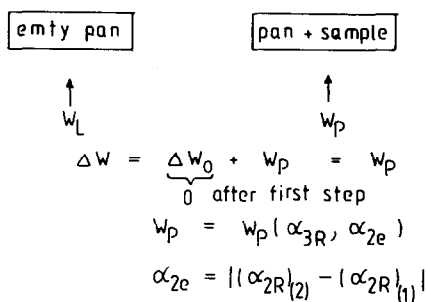


Fig. 2 Zero setting procedure

### 3. MATHEMATICAL SOLUTION

#### 3.1. SPECIFIC HEAT

The parabolical initial-boundary condition problem of the third kind (see fig. 1) was solved by analytical and numerical methods.

The solution  $W(T)$  is the heating flux for heating the sample system with the heating rate  $K_1$  from the constant initial temperature  $T_A$  to the constant final temperature  $T_E$ .  $W(T_A)$  and  $W(T_E)$  are the time independent heat fluxes for maintaining the constant temperatures  $T_A$  and  $T_E$ .

The deviations between analytical and numerical solutions are smaller than  $10^{-4}\%$  in temperature ranges where phase transitions are absent.

With the definitions

$$\begin{array}{ll}
 D_2 = \text{thickness of sample} & D_1 = \text{effective thickness of pan} \\
 \lambda_2 = \text{heat conductivity of sample} & \lambda_1 = \text{heat conductivity of pan} \\
 \rho_2 = \text{density of sample} & \rho_1 = \text{density of pan} \\
 c_2 = \text{specific heat of sample} & c_1 = \text{specific heat of pan}
 \end{array}$$

and the abbreviations

$$\begin{array}{ll}
 a_1 = \lambda_1 / (\rho_1 c_1) & a_2 = \lambda_2 / (\rho_2 c_2) \\
 h_1 = \alpha_{1R} / \lambda_1 & h_2 = \alpha_{2R} / \lambda_2 \\
 h_{13} = \alpha_{3R} / \lambda_1 & h_{23} = \alpha_{3R} / \lambda_2
 \end{array}$$

$$a = h_2 h_{23} D_2 + h_2 + h_{23}$$

$$b = D_2 + h_2 h_{23} D_2^3 / 6 + (h_2 + h_{23}) D_2^2 / 2$$

$$c = 1 + h_2 D_2$$

$$d = h_2 h_{23} D_2 + h_2 + h_{23}$$

$$e = D_2^2 / 2 + h_2 D_2^3 / 6$$

$$g = h_1 h_{13} D_1 + h_1 + h_{13}$$

$$r = D_1 + h_1 h_{13} D_1^3 / 6 + (h_1 + h_{13}) D_1^2 / 2$$

$$s = a_2 / a_1$$

$$A = g \cdot a - h_{13} h_{23} c (1 + h_1 D_1)$$

$$E = (1/a_2) [gb - h_{13} h_{23} (1 + h_1 D_1) e] +$$

$$(1/a_1 [dr - h_{13} h_{23} c (D_1^2 / 2 + h_1 D_1^3 / 6)])$$

$$z = (T - T_A) / K_1 - E/A$$

the solutions are

$$W(T_A) = \frac{\lambda_2 T_A}{(1/h_{23}) (1 + h_{13}/h_1 + D_1 h_{13}) + 1/h_2 + D_2} \quad (1)$$

$$W(T_E) = \frac{\lambda_2 T_E}{(1/h_{23}) (1 + h_{13}(h_1 + D_1 h_{13})) + 1/h_2 + D_2} \quad (2)$$

$$W(T) = - (\lambda_1 h_1 K_1 / A) [(h_{13} h_{23} c - h_{13} a) z + (1/a_2) (h_{13} h_{23} (e + s c D_1^2 / 2) - h_{13} b - s d (D_1 + h_{13} D_1^2 / 2))] - (1/\lambda_1) W(T_A) \quad (3)$$

The expression  $W(T)$  is the equation of a straight line with the slope  $m_B$  and with the ordinate section  $W_0$ , therefore

$$W_B(T) = W_0 + m_B T \quad (4)$$

$W_0$  and  $m_B$  are functions of the thermophysical and geometrical parameters and of the heating rate and the heat-transfer coefficients according (3).

For  $D_1 = 0$  and  $\alpha_{1R} \rightarrow \infty$  the pure heat flux  $W_p$  into the sample without pan is obtained, which is necessary for the application of the procedure of zero setting (see 2.3.).

In the following this procedure will be used consequently.

Figure 3 shows a typical calculated heat flux curve.

For ideal heat contacts ( $\alpha_{3R} \rightarrow \infty$ ) and adiabatic behaviour ( $\alpha_{2e} \rightarrow 0$ ) no temperature dependence is visible and the signal has the ideal value (Fig. 3 (i))

$$W = W_s = D_2 \int_2 c_2 K_1$$

An example of ideal phase transition with the transition temperature  $T_g$  is shown, too.

For real heat contacts ( $\alpha_{3R} < \infty$ ) and heat dissipation to the isoperibole surroundings ( $\alpha_{2e} > 0$ ) the quasistationary heat flux  $W(T)$  is temperature dependant (see Fig. 3 (ii)) and the signal  $\Delta W = W_*$  is smaller than the ideal signal  $W_s$ .

The isothermal heat fluxes  $W(T_A)$  and  $W(T_E)$  do not disappear under real conditions, where

$$W(T_E) > W(T_A)$$

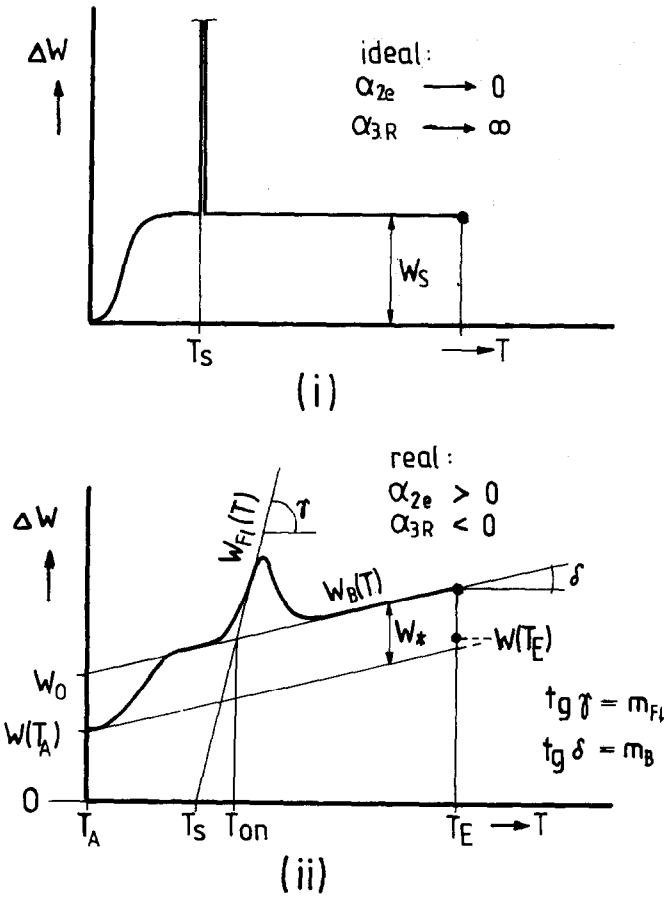


Fig. 3 Graphical representation of mathematical solution

The deviation of the real signal  $W_*$  from the ideal signal  $W_S$  is defined as

$$F = \frac{|W_* - W_S|}{W_S}$$

and the difference between  $W(T_A)$  and  $W(T_E)$  related to  $W_S$  is defined as isothermal distance



$$V = \frac{W(T_E) - W(T_A)}{W_S}$$

In figure 4 the deviation  $F$  is shown as a function of isothermal distance  $V$  at different values of  $\alpha_{2e}$  and  $\alpha_{3R}$ .

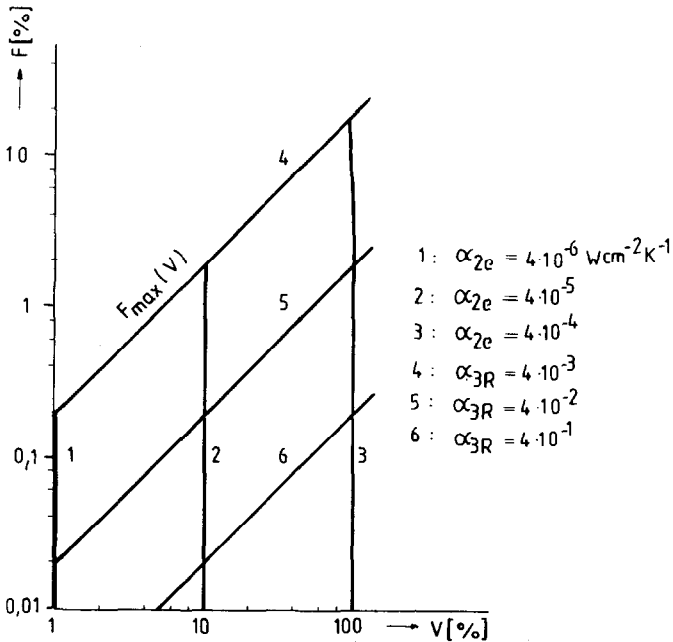


Fig. 4 Deviation  $F$  between true and real calorimetrical signal as a function of isothermal distance

It is seen that the deviation  $F$  can be hold small (for example  $F \leq 0,2 \%$ ) for two conditions:

1. strong dissipation of heat (curve 3) with very good heat contacts (curve 6)

2. small dissipation of heat (curve 1) with any heat contact (for example curve 4 which represents very poor heat contact)

On the other side the deviation  $F$  can be very great if there are very poor heat contacts (curve 4) and great heat dissipation (curve 3), therefore  $F = 17\%$  for the crossing point of the curves 3 and 4 will be reached. It can be also seen that only the maximal values  $F_{\max}(V)$  caused by maximal heat resistances ( $\alpha_{3R} = 4 \cdot 10^{-3} \text{ Wcm}^{-2}\text{K}^{-1}$ ) have a unambiguous relation to the isothermal distance  $V$  (here represented by curve 4). For smaller heat resistances there is no such unambiguous relation between  $F$  and  $V$ .

An influence of the heat conductivity and of the heating rate on the value of  $F$  was not found.

### 3.2. TRANSITION HEATS AND TRANSITION TEMPERATURES

The true value of specific transition energy is generally not influenced by the thermophysical and geometrical properties of sample neither by the heating rate and the heat-transfer coefficients. Only the shape of the DSC-signal is modified. Figure 3 (ii) shows schematically and figure 5 precisely the deformation of the originally very sharp transition mainly caused by small values of  $\alpha_{3R}$ . Under the supposition of using the zero setting procedure (see 2.3.) the slope at the point of inflexion of the ascending parts of the transition curves is in every case equal to  $\alpha_{3R}$  independently from heat capacity, specific transition energy, heat conductivity and heating rate (see fig. 5) for small heat capacities especially.

The tangent at the point of inflexion cuts the zero line exactly in the true transition temperature  $T_g$ , which is here  $505^\circ\text{C}$ .

It is remarkable that for smaller values of heat conductivity the slope decreases more or less near the maximum of the  $\Delta W$ -signal (see curves 2, 4, 7, 8 and especially curve 5).

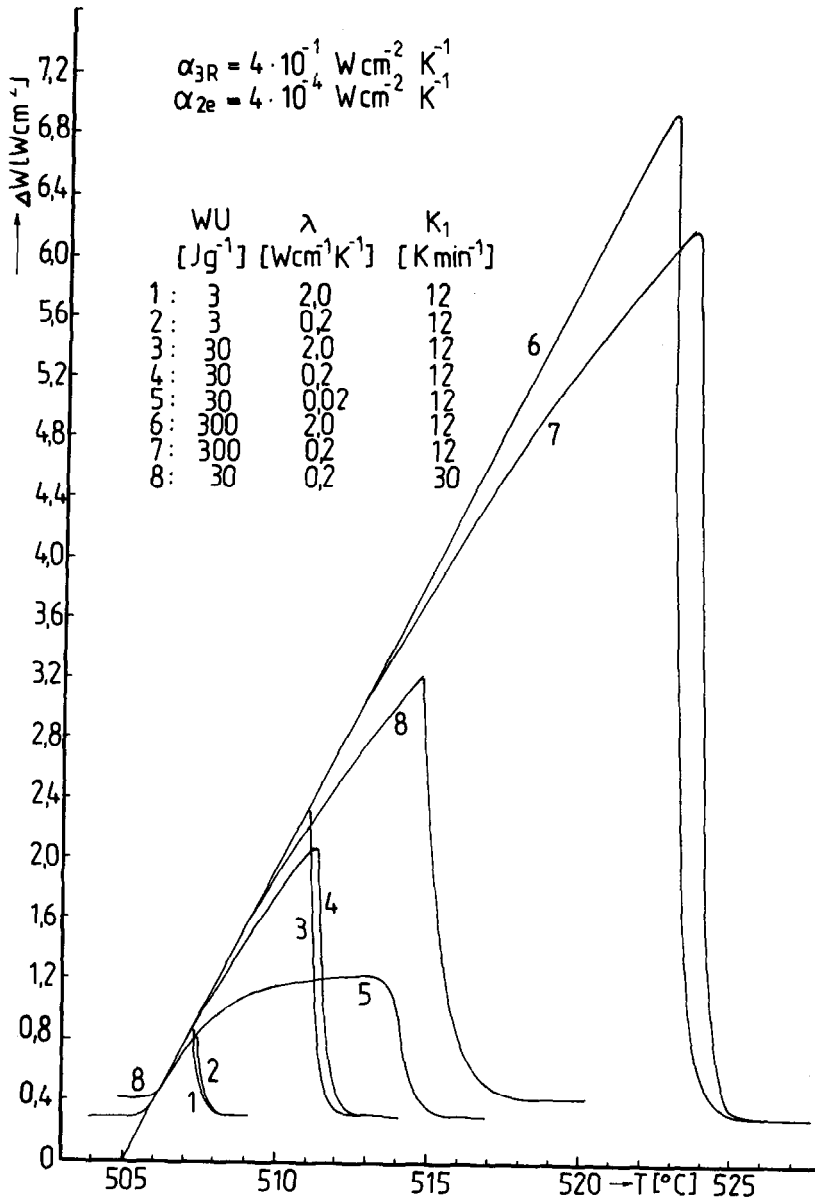


Fig. 5 Numerically calculated transition shapes under different conditions

In figure 6 the relations between the true transition temperature  $T_s$ , the extrapolated onset temperatures  $T_{on}$  and different values of specific heat, density, heating rate and heat-transfer coefficients are presented, because the height of every baseline A, B, or C is proportional to specific heat, density, heating rate and the heat-transfer coefficient  $\alpha_{2e}$ .

The higher the baseline the greater the difference between  $T_s$  and  $T_{on}$ .

The smaller the value of  $\alpha_{3R}$  the smaller the slope and therefore the greater the shifting of  $T_{on}$  to higher temperatures. (See the slopes labelled by I and II).

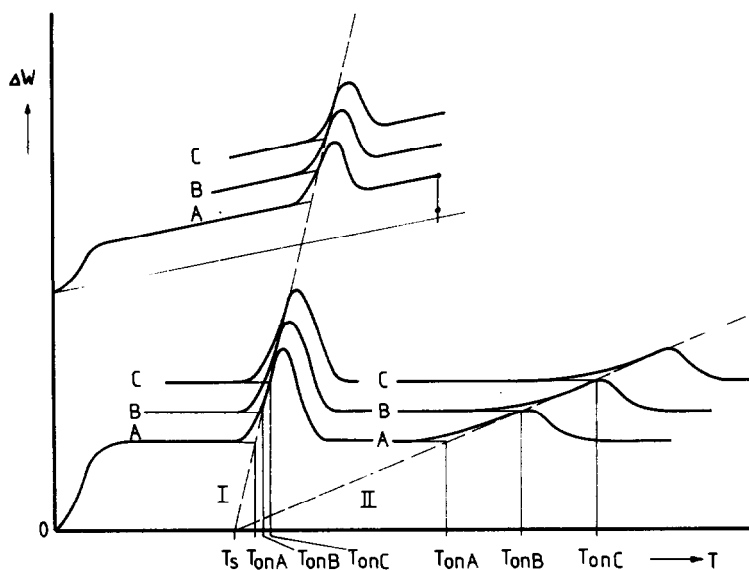


Fig. 6 Relations between true transition temperature, onset temperatures, thermophysical properties, heating rates and heat-transfer coefficients (schematically)

### 3.3. THEORETICAL DETERMINATION OF THE TRUE TRANSITION TEMPERATURE $T_s$

#### 3.3.1. ZERO-CUT-METHOD

The equation for the tangent at the point of inflexion has in every case the form (see fig. 3)

$$W_{FL}(T) = 3R (T - T_s) \quad (5)$$

For  $W_{FL}(T) = 0$  follows

$$T = T_s$$

Thus the crossing point of (5) with the axis  $W = 0$  represents the true transition temperatur  $T_s$  for all cases.

#### 3.3.2. CROSSING-POINT-METHOD

The crossing point of the baseline (4) with the tangent (5) at the inflexion point is the onset temperatur  $T_{on}$ .

From (4) and (5) follows

$$T_{on} = \frac{m_{FL} \cdot T_s + W_o}{m_{FL} - m_B} \quad \text{or} \quad (6)$$

$$T_s = \frac{T_{on} (m_{FL} - m_B) - W_o}{m_{FL}} \quad (7)$$

Every DSC-measurement with a phase transition produces  $T_{on}$ ,  $m_{FL}$ ,  $m_B$  and  $W_o$ . Thus it is possible to determinate  $T_s$  also in this manner.

## 4. CONCLUSION

As shown in this work, the DSC-measurement produces neither true calorimetrical signals for determination of specific heat nor the true transition temperatures.

Whereas it is impossible to reconstruct the true calorimetrical signal from the measured signal, the determination of the true transition temperatur from the measured values can be accomplished at the same measurement independently from mass, heat capacity, heating rate and heat-transfer coefficients. This has the consequence of a remarkable simplification of the determination of transition temperatures and of a essential improvement of precision at the same time. Up to now it has

been usual to measure the onset temperature at very many masses and heating rates for determination the true transition temperature by extrapolating the mass and the heating rate towards zero.

With the theoretically derived methods of the zero-cut-method and the crossing-point-method it is easy to develop a suitable software for precise und quick calculation of the true transition temperatures instead of using the time consuming onset methods.

## 5. REFERENCES

- [1] W. Poeßnecker, Journal of Thermal Analysis, Vol. 36 (1990), 1123 - 1139
- [2] Hemminger/Höhne, Grundlagen der Kalorimetrie, Verlag Chemie, Weinheim, New York, 1979