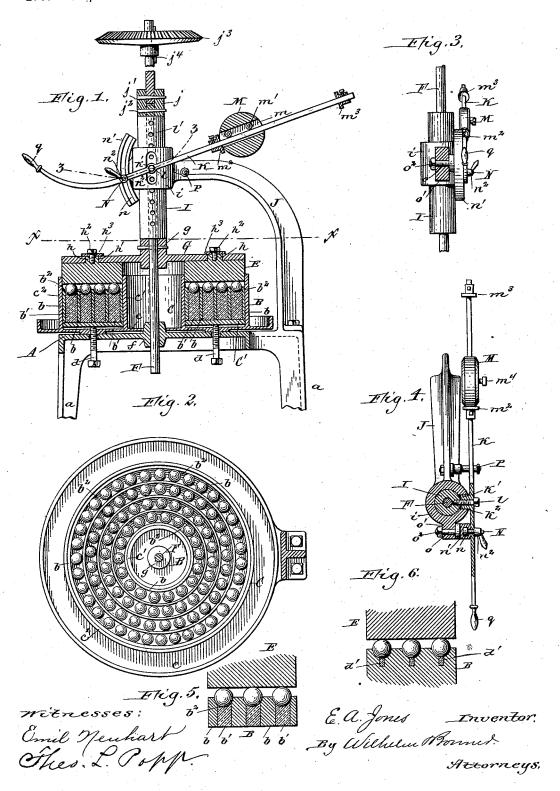
E. A. JONES.

MACHINE FOR GRINDING METAL BALLS.

No. 455,879.

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UNITED STATES PATENT OFFICE.

EDWARD A. JONES, OF TONAWANDA, NEW YORK.

MACHINE FOR GRINDING METAL BALLS.

SPECIFICATION forming part of Letters Patent No. 455,879, dated July 14, 1891.

Application filed January 19, 1891. Serial No. 378,238. (No model.)

To all whom it may concern:

Be it known that I, EDWARD A. JONES, a subject of the Queen of Great Britain, residing at Tonawanda, in the county of Erie and State of New York, have invented a new and useful Improvement in Machines for Grinding Metal Balls, of which the following is a specification.

This invention relates to a ball-grinding 10 machine in which metallic balls suitable for use in ball-bearings are ground and polished between two disks.

My invention has for its objects to improve the construction of the lower grinding-disk, 15 so as to increase the effectiveness of its grinding-surface, and also to provide an automatic device for depressing and raising the upper grinding-disk.

In the accompanying drawings, Figure 1 is 20 a sectional elevation of my improved ballgrinding machine. Fig. 2 is a horizontal section in line x x, Fig. 1. Fig. 3 is a fragmentary front elevation of the raising and depressing mechanism. Fig. 4 is a horizontal section in line zz, Fig. 1. Figs. 5 and 6 are fragmentary vertical sections showing modified forms of the lower grinding-disk.

Like letters of reference refer to like parts

in the several figures.

A represents the bed-plate of the machine,

which is supported on legs a a.

B and E represent the disks between which the balls are ground and polished. The lower disk B consists of a series of metallic divis-35 ion-rings b, arranged concentrically and separated from each other by annular spaces, which are filled with emery or other suitable abrading material. The emery is compacted in the annular spaces, so as to form solid 40 rings or fillings b' between the metallic rings. The emery fillings are provided in their upper sides with annular grooves b^2 , of halfround form, in which the rows of balls are seated. The emery rings may be made of 45 greater or less width, but should be wide enough to form effective abrading-surfaces for the balls, whereby the latter are ground and polished when rolled between the disks. The metallic division-rings extend above the em-50 ery rings and form the sides of the grooves

which receive the balls. The division-rings are somewhat harder than the emery in the bottom of the grooves, which causes the grooves to wear downwardly. This prevents the balls from moving laterally or radially with refer- 55 ence to the center of the disks and wearing the grooves wide, which would cause the balls to be ground irregularly.

C represents an annular frame or trough which contains the division and emery rings 60 and an annular supporting-plate C', arranged on the bottom plate c of the trough between the inner and outer walls c' c^2 thereof, whereby the plate C' is guided vertically. This frame or trough rests upon the bed-plate and is pro- 65 vided with adjusting-screws d, arranged in the base-plate of the frame and bearing against the lower side of the supporting-plate, whereby the latter and the grinding-disk resting thereon are adjusted vertically. The adjust- 70 ing-screws extend through enlarged openings in the bed-plate, which permits the grindingdisk to be adjusted from below and also holds the same against rotary movement.

In the construction of the lower grinding- 75 ring represented in Fig. 1 the emery rings are equal in width to the balls which are being ground, so that a large portion of each ball is presented to the grinding-surface of

the emery ring.

In the modified construction represented in Fig. 5 the emery rings are somewhat narrower than the balls, and the metallic division-rings extend underneath the balls, so as to form a portion of the bottom support for 85 the balls. This arrangement causes the division-rings to retard both the lateral and downward wearing of the grooves and requires less frequent renewal of the emery rings.

In the modified construction represented in Fig. 6 the lower grinding-disk consists of one solid metallicdisk having the annular grooves for the balls cut in its upper surface, and each of the grooves being provided in its bot- 95 tom with a channel d', which is filled with emery or similar abrading material. In each case an abrasive bearing-surface is provided for the lower portion of the balls, which accelerates the grinding and polishing of the 100 latter and retards the lateral wearing of the

Heretofore the emery has been introduced between the grinding-disks in a powdered form for the purpose of grinding the balls; but the centrifugal action of the rotating upper disk and the balls caused the powdered emery to work over the sides of the disks and required frequent renewal of the emery.

10 By forming a bed of solid emery to support the balls a constant abrading-surface is furnished, which overcomes this difficulty and causes the balls to be ground more uniformly.

The upper grinding-disk E consists of one solid ring which extends downwardly between the walls of the annular frame and bears upon

the balls.

F represents a vertical shaft whereby a rotary motion is imparted to the upper grinding-disk. This shaft is arranged within both grinding-disks and axially in line therewith and is journaled near its lower end in a bearing f, formed centrally in the bed-plate of the machine.

25 G represents a supporting-plate whereby the upper disk is secured to the shaft. The central portion of this plate is provided with a hub g, which is secured to the shaft. The upper disk is provided on its upper side with 30 bosses or lugs h, which enter openings h', formed in the upper supporting-plate, and the disk is secured to the latter by screw-bolts h², passing through washers h³, arranged on the supporting-plate and entering the bosses

35 of the upper disk.

The upper end of the shaft is journaled in a sleeve I, arranged to slide vertically in a collar i, formed at the upper end of an overhanging arm J, secured to one side of the 40 bed-plate. The vertical shaft is preferably made in two sections for convenience in disconnecting the parts, the adjoining ends being connected by means of a coupling-collar j, secured to the ends of the shaft-sections by means of transverse pins j' j^2 . The sleeve I bears with its ends against the hub of the upper disk and the coupling-collar, whereby lengthwise movement of the sleeve on the shaft is prevented. The upper portion of the 50 shaft is provided with a bevel-gear j3, for the purpose of transmitting motion to the upper grinding-disk. The gear-wheel is secured to the shaft by means of a key or spline j^4 , which enables the shaft to move lengthwise 55 through the gear-wheel, but compels them to rotate together.

K represents a lever whereby the upper grinding-disk can be depressed for grinding the balls or raised for the purpose of removing the balls. This lever is provided between its ends with a boss k', which is arranged in a vertical slot k², formed on one side of the collar i of the arm J, and is pivotally secured to the side of the sleeve I by means of a bolt lengaging with one of a vertical series of openings l', formed on the side of said sleeve.

M represents a sliding weight arranged on the rear arm of the lever, whereby the rear arm is depressed. This weight is provided with an opening m, through which the lever 7c passes and which contains rollers m', which rest upon the lever and enable the weight to slide easily. The movement of the sliding weight on the lever is limited by adjustable stops m^2 m^3 , secured to the rear arm of the 75 lever and arranged on opposite sides of the

weight.

N represents a fulcrum-bolt, which serves as the fulcrum of the lever K when the latter is in the position for depressing the upper 80 grinding-disk. This bolt is arranged with its head in an undercut groove n, formed in an adjusting-bar n', and passes through an opening formed in the outer arm of the lever. The lever and adjusting-bar are pivotally secured 85 together by a clamping-nut n^2 , applied to the screw-threaded end of the fulcrum-bolt and bearing against the side of the lever. The adjusting-bar is provided on one side with a screw-threaded shank o, which passes through 90 an opening formed in a lug o' on the front side of the collar i, and is secured thereto by a clamping-nut o², applied to the end of the shank. The shank and clamping nut of the adjusting-bar enable the latter to be adjusted 95 to the position of the fulcrum-bolt N in changing the fulcrum of the lever K.

P represents a fulcrum pin or roller secured to the side of the overhanging arm underneath the rear arm of the lever K. This pin or roller serves as a fulcrum for the latter when it is desired to raise the upper disk for the purpose of removing the balls and placing a new supply upon the lower disk. In the position of the parts represented in Fig. 105 1 the lever is fulcrumed on the bolt N, which causes the sleeve carrying the shaft and upper grinding-disk to be depressed, owing to the pressure of the weight at the opposite end of the lever. As the upper disk wears 110 and becomes lighter the weight is moved toward the outer end of the lever by adjusting the stop m^2 to increase its leverage and compensate for the loss in the pressure resulting from the wearing of the upper disk. During 115 the downward movement of the lever, as the upper disk wears away, its fulcrum-bolt is permitted to move outwardly by a slight turning of the adjusting-bar on its pivot. The groove in the adjusting-bar permits the 120 fulcrum-bolt to be adjusted vertically in accordance with the thickness of the upper grinding-disk. When it is desired to raise the upper disk, the fulcrum-bolt is loosened, which permits the front arm of the lever to 125 rise and the rear arm to descend until the latter rests upon the fulcrum pin or roller P. Upon lowering the rear arm of the lever slightly below a horizontal position the weight M slides toward the outer end of the rear arm 130 by gravity, and thereby increases the leverage of the latter. This increased leverage

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causes the rear arm to continue its downward movement and swing on the roller P as a fulcrum, thereby raising the front arm of the lever and lifting the upper disk connected 5 therewith. The adjusting-bar is preferably curved, so as to permit the fulcrum-bolt to move freely in the groove of the adjustingbar when the lever swings on the roller P as a fulcrum. In this manner the weight M is 10 utilized either to depress or raise the upper grinding-disk, thereby enabling the machine to be manipulated very easily. When it is desired to again depress the upper disk, the front arm of the lever is depressed by means 15 of the handle q at the front end until the upper disk bears upon the balls and the rear arm of the lever is raised from the fulcrumroller. Upon tightening the bolt N on the adjusting-bar said bolt again acts as the 20 fulcrum of the lever, which causes the weight to depress the upper disk. By alternately locating the fulcrum on the bolt N and roller P it acts alternately in the capacity of a first and second class lever. The in-25 stant the rear arm of the lever is raised above a horizontal position the weight slides inwardly toward the fulcrum, thereby decreasing the leverage. In this manner a small leverage and a correspondingly light 30 downward pressure can be applied to the balls in addition to the weight of the upper disk, while a larger leverage is used when it is desired to raise the upper disk. This change in leverage is automatically effected 35 by the sliding of the weight M when the rear arm of the lever rises above or falls below a horizontal position. If desired, the weight may be rigidly secured in place after adjustment by means of a set-screw m^4 .

The boss of the lever and its pivot-bolt l are free to move vertically with the sleeve I, owing to the slot k^2 . The pivot-bolt l is placed in one of the higher openings l' as the upper disk becomes worn, so as to retain its rela-

45 tive position to the fulcra of the lever.

I claim as my invention-

1. A ball-grinding disk having its workingface composed of concentric metallic rings and intermediate abrasive fillings arranged 50 between the metallic rings and depressed below the same, thereby forming concentric grooves having metallic sides and abrasive bottoms, substantially as set forth.

2. In a ball-grinding machine, the combi-55 nation, with the upper grinding-disk, of a lower grinding-disk having its upper face provided with annular grooves having metallic sides and abrasive fillings forming the bottoms of the grooves, whereby the balls 60 wear the grooves downwardly into the abrasive fillings and are restrained radially by the metallic sides of the grooves, substantially as set forth.

3. In a disk for ball-grinding machines, the 65 combination, with the series of grindingnular grooves, and division-rings arranged between said grinding-rings, of a plate supporting said grinding-rings and division-rings and adjusting-screws engaging against said 70 plate, whereby the disk is adjusted, substan-

tially as set forth.

4. In a disk for ball-grinding machines, the combination, with the series of grindingrings provided in their upper sides with an- 75 nular grooves, and division-rings arranged between said grinding-rings, of a plate supporting said grinding-rings and division-rings, adjusting-screws bearing against said supporting-plate, and an annular frame contain- So ing said rings and supporting-plate, substantially as set forth.

5. In a ball-grinding machine, the combination, with a grinding-disk and its support, of a weighted lever pivotally secured to said \$5 support and connected with said grindingdisk, whereby the latter is moved vertically,

substantially as set forth.

6. The combination, with the movable grinding-disk and its supporting-frame, of a 90 lever connected with said disk and having fulcra arranged on opposite sides of its connection with said disk, and a weight attached to the lever, substantially as set forth.

7. The combination, with the movable 95 grinding-disk and its supporting-frame, of a lever attached to said disk, and a sliding weight arranged on said lever, whereby the leverage is automatically increased or decreased, substantially as set forth.

8. The combination, with the movable grinding-disk and its supporting-frame, of a lever connected with said disk and frame near one end, a weight arranged to slide upon the opposite end of said lever, and stops 105 whereby the sliding movement of said weight is limited, substantially as set forth.

9. The combination, with the movable grinding-disk, a movable sleeve connected with said disk, and a supporting-frame in 110 which said sleeve is guided, of a lever connected with said sleeve, a pivoted support on said frame for one arm of said lever, and a weight arranged on the opposite arm of said lever, substantially as set forth.

10. The combination, with the movable grinding-disk, a movable sleeve connected with said disk, and a frame supporting said sleeve, of a lever attached to said sleeve, a pivoted bar to which one arm of said lever 120 is adjustably attached, and a weight arranged on the opposite arm of said lever, substan-

tially as set forth.

11. The combination, with the bed-plate, the shaft, and the rotating disk secured to said 125 shaft, of a sleeve surrounding said shaft and provided with a vertical series of openings, an arm secured to said plate and provided with a collar surrounding said sleeve, a lever pivotally supported by a bolt entering one of 130 the openings in the sleeve, an adjusting-bar rings provided in their upper sides with an- pivotally supported on said arm, a bolt capa-

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ble of adjustment in said bar and adapted to serve as a fulcrum for the lever on one side of the sleeve, a pin or roller attached to the arm and adapted to serve as a fulcrum for the lever on the opposite side of the sleeve, and a weight attached to said lever, substantially as set forth.

Witness my hand this 13th day of January, 1891.

EDWARD A. JONES.

Witnesses:
THEO. L. POPP,
EMIL NEUHART.